

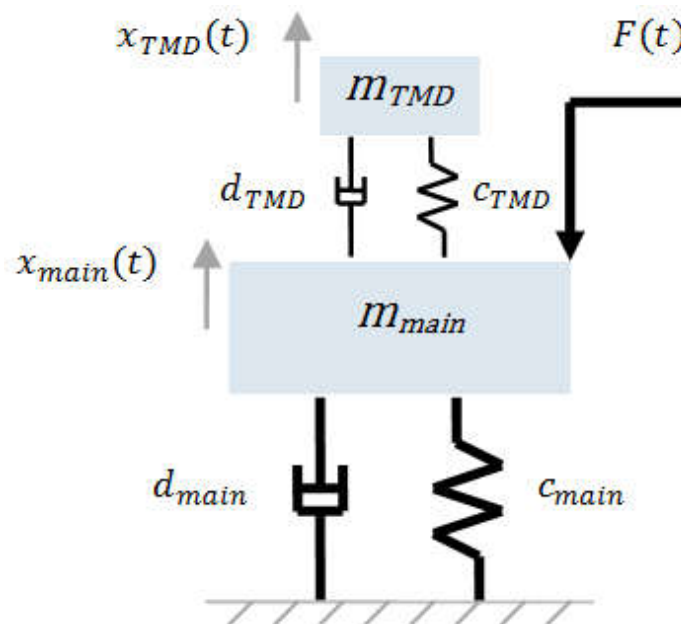
Tuned Mass Damper Design for Attenuating Vibration

▼ Introduction

A mass-spring-damper is disturbed by a force that resonates at the natural frequency of the system.

This application calculates the optimum spring and damping constant of a parasitic tuned-mass damper that the minimizes the vibration of the system.

The vibration of system with and without the tuned mass-spring-damper is viewed as a frequency response, time-domain simulation and power spectrum.



> restart : with(DynamicSystems) : with(plots) : with(SignalProcessing) :

> params := $[m_1 = 1.764 \cdot 10^5, k_1 = 3.45 \cdot 10^7, b_1 = 1.531 \cdot 10^5, m_2 = 8165]$:

▼ Derive Expressions for the Optimum Spring and Damping Constant of the Tuned Mass Damper

Mass ratio

$$> \mu := \frac{m_2}{m_1} :$$

Natural frequency of tuned mass damper

$$> \omega_2 := \sqrt{\frac{k_{2_calc}}{m_2}} :$$

Natural frequency of main system

$$> \omega_1 := \sqrt{\frac{k_1}{m_1}} :$$

Hence the natural frequency in rad s⁻¹ is

$$> \text{eval}(\omega_1, \text{params})$$

13.98492872

Ratio of natural frequencies

$$> \alpha := \frac{\omega_2}{\omega_1} :$$

Optimum ratio of natural frequencies

$$> \alpha_{opt} := \frac{1}{1 + \mu} :$$

Hence the optimum spring constant of the tuned mass-spring-damper

$$> k_{2_calc} := \text{solve}(\alpha = \alpha_{opt}, k_{2_calc})$$

$$k_{2_calc} := \frac{m_1 k_1 m_2}{(m_1 + m_2)^2}$$

Damping Ratio

$$> z := \frac{b_{2_calc}}{2 m_2 \omega_2} :$$

Optimum damping ratio

$$> z_{opt} := \sqrt{\frac{3\mu}{8(1+\mu)^3}} :$$

Hence the optimum damping constant of the tuned mass-spring-damper

$$> b_{2_calc} := \text{solve}(z = z_{opt}, b_{2_calc})$$

$$b_{2_calc} := \frac{1}{2} \sqrt{6} \sqrt{\frac{m_2}{m_1 \left(1 + \frac{m_2}{m_1}\right)^3}} m_2 \sqrt{\frac{m_1 k_1}{(m_1 + m_2)^2}}$$

> $k_{2_calc} := \text{eval}(k_{2_calc}, \text{params});$

$$k_{2_calc} := 1.45873086110^6$$

> $b_{2_calc} := \text{evalf}(\text{eval}(b_{2_calc}, \text{params}));$

$$b_{2_calc} := 26869.77094$$

Full list of parameters

> $\text{params}_{\text{tuned}} := [m_1 = 1.764 \cdot 10^5, k_1 = 3.45 \cdot 10^7, b_1 = 1.531 \cdot 10^5, m_2 = 8165, k_2 = k_{2_calc}, b_2 = b_{2_calc}];$

> $\text{params}_{\text{nottuned}} := [m_1 = 1.764 \cdot 10^5, k_1 = 3.45 \cdot 10^7, b_1 = 1.531 \cdot 10^5, m_2 = 0, k_2 = k_{2_calc}, b_2 = b_{2_calc}];$

▼ Equations of Motion for the Entire System

Equation of motion for the whole system

> $\text{de} := m_2 \left(\frac{d^2}{dt^2} x_2(t) \right) = -k_2 (x_2(t) - x_1(t)) - b_2 \left(\frac{d}{dt} x_2(t) - \frac{d}{dt} x_1(t) \right),$

$m_1 \left(\frac{d^2}{dt^2} x_1(t) \right) = -k_1 x_1(t) - b_1 \left(\frac{d}{dt} x_1(t) \right) - k_2 (x_1(t) - x_2(t)) - b_2 \left(\frac{d}{dt} x_1(t) - \frac{d}{dt} x_2(t) \right) + F(t) :$

$\text{ic} := x_1(0) = 0, D(x_1)(0) = 0, x_2(0) = 0, D(x_2)(0) = 0 :$

> $\text{sys} := \text{DiffEquation}([\text{de}, [F(t)], [x_1(t)]]):$

▼ Frequency Response

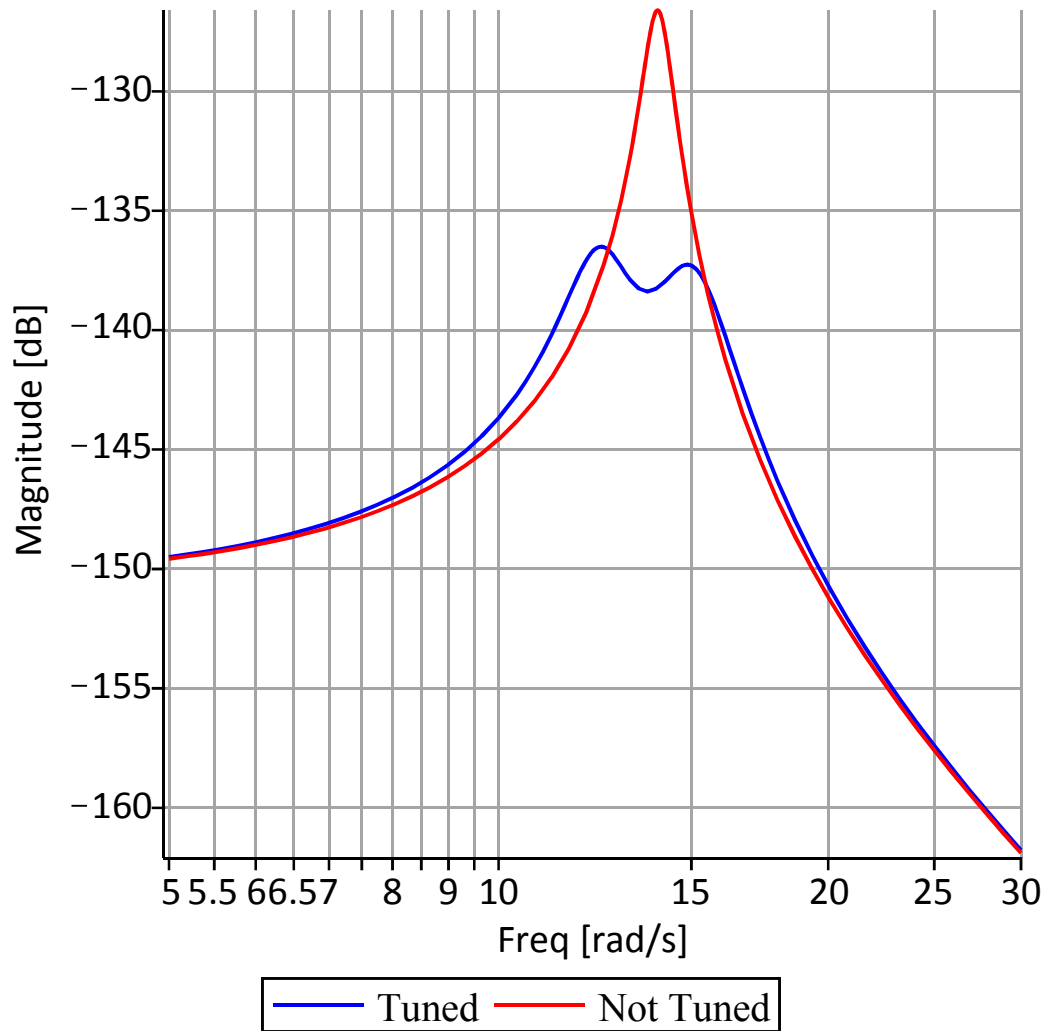
Response with Tuned Mass Damper

> $\text{p1} := \text{MagnitudePlot}(\text{sys}, \text{range} = 5..30, \text{parameters} = \text{params}_{\text{tuned}}, \text{color} = \text{blue}, \text{legend} = \text{"Tuned"}) :$

Response with no Tuned Mass Damper

> $\text{p2} := \text{MagnitudePlot}(\text{sys}, \text{range} = 5..30, \text{parameters} = \text{params}_{\text{nottuned}}, \text{color} = \text{red}, \text{axesfont} = [\text{Calibri}], \text{labelfont} = [\text{Calibri}], \text{legend} = \text{"Not Tuned"}) :$

> $\text{display}(\text{p1}, \text{p2})$



▼ Dynamic Response

Assuming that the main system is perturbed at its natural frequency

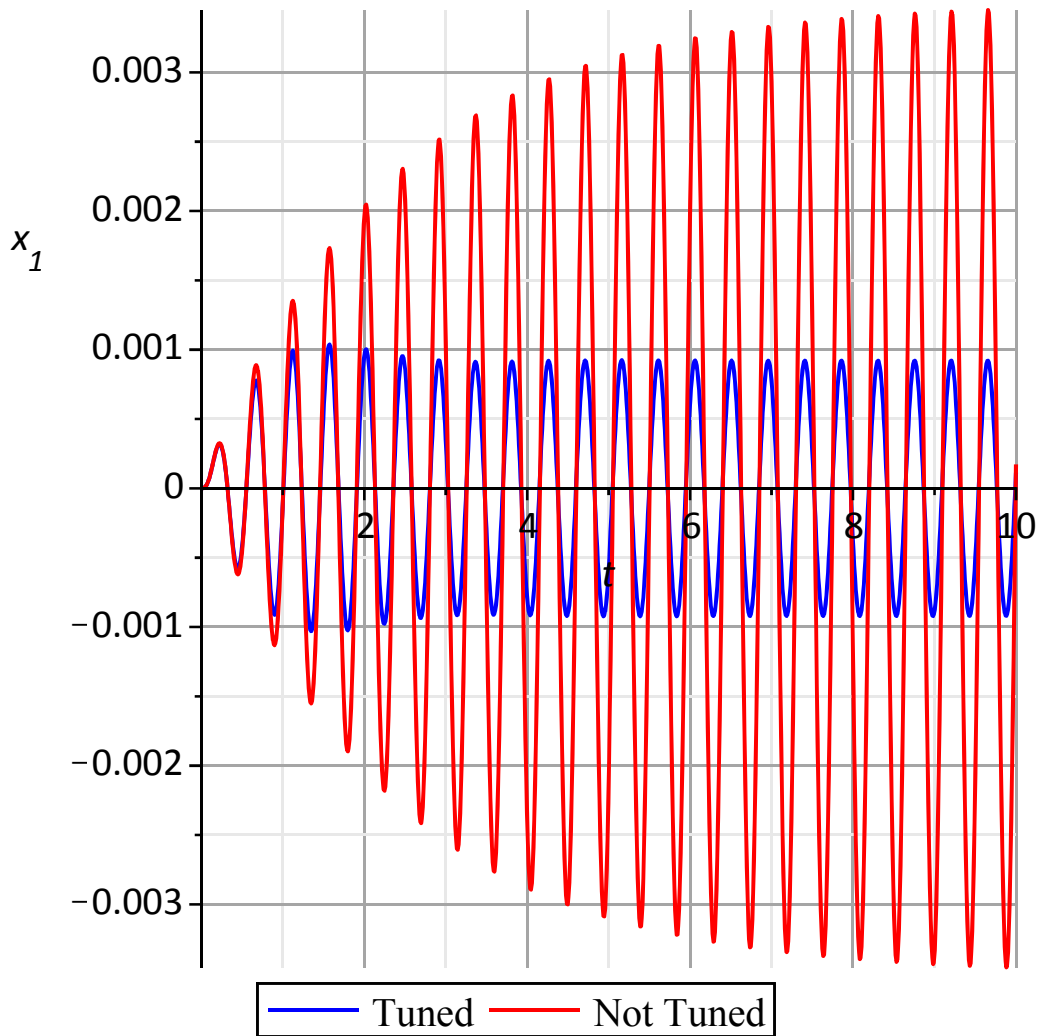
> $f := \text{eval}(\omega_1, \text{params})$

$f := 13.98492872$

> $p1 := \text{ResponsePlot}(\text{sys}, 7500 \sin(f \cdot t), \text{parameters} = \text{params}_{\text{tuned}}, \text{color} = \text{blue}, \text{numpoints} = 2^{10}, \text{legend} = \text{"Tuned"}) :$

> $p2 := \text{ResponsePlot}(\text{sys}, 7500 \sin(ft), \text{parameters} = \text{params}_{\text{nottuned}}, \text{color} = \text{red}, \text{numpoints} = 2^{10}, \text{legend} = \text{"Not Tuned"}) :$

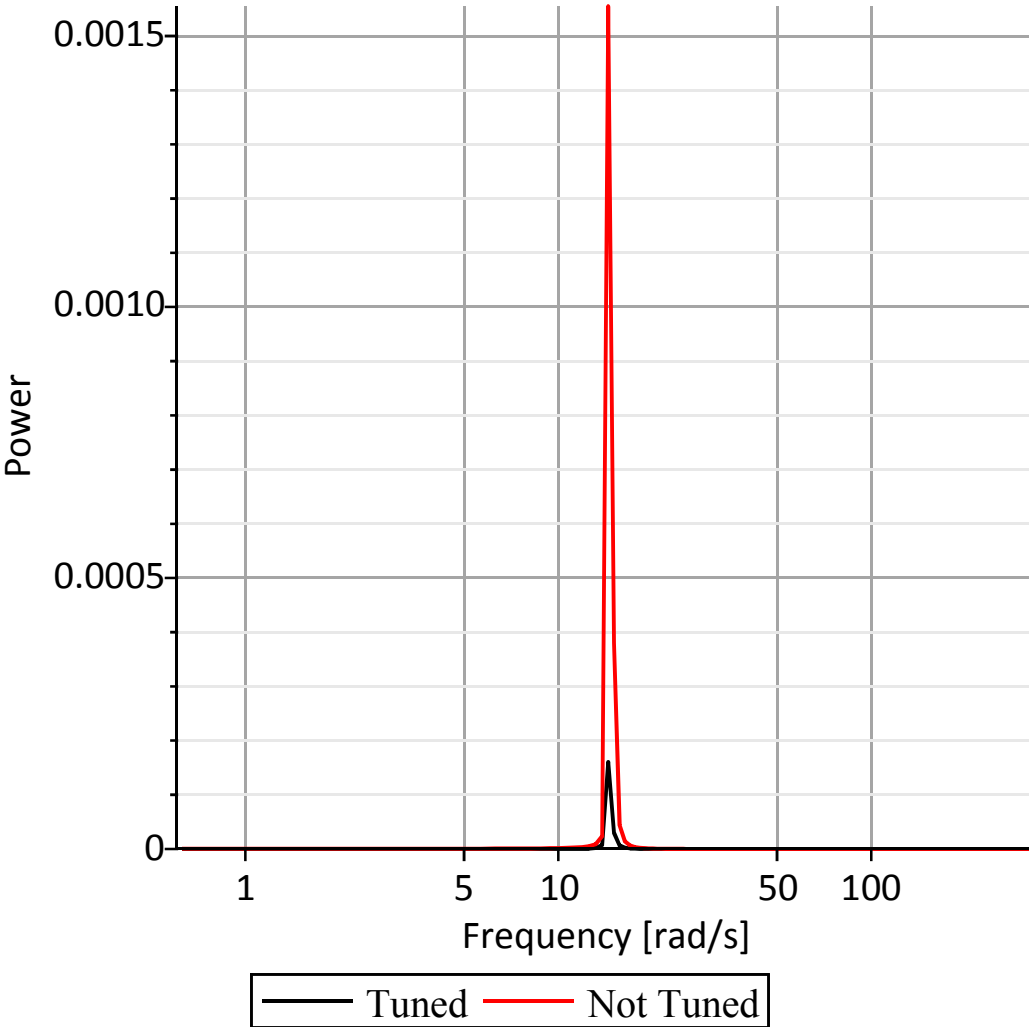
> $\text{display}(p1, p2, \text{axesfont} = [\text{Calibri}], \text{labelfont} = [\text{Calibri}], \text{axesfont} = [\text{Calibri}], \text{gridlines})$



▼ Power Spectrum

- > tunedResponseData := plottools:getdata(p1)[3]:
- > notTunedResponseData := plottools:getdata(p2)[3]:
- > samplingRate := $\frac{1}{\text{tunedResponseData}[2,1] - \text{tunedResponseData}[1,1]}$:
- > psTuned := PowerSpectrum(FFT(tunedResponseData[.,2])):
- > psNotTuned := PowerSpectrum(FFT(notTunedResponseData[.,2])):
- > psPlot1 := pointplot($\left[\text{seq} \left(\left[\frac{i \cdot \text{samplingRate}}{2^{10}} \cdot 2 \cdot \text{Pi}, \text{psTuned}[i] \right], i = 1 .. \frac{2^{10}}{2} \right) \right]$, connect = true, legend = "Tuned", gridlines):
- psPlot2 := pointplot($\left[\text{seq} \left(\left[\frac{i \cdot \text{samplingRate}}{2^{10}} \cdot 2 \cdot \text{Pi}, \text{psNotTuned}[i] \right], i = 1 .. \frac{2^{10}}{2} \right) \right]$, connect = true, color = red, legend = "Not Tuned"):
- > display(psPlot1, psPlot2, axis[1] = [mode = log], axesfont = [Calibri], labels = ["Frequency [rad/s]"],

"Power"], labeldirections = [horizontal, vertical], labelfont = [Calibri])



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